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TABLE  
GIVING AT INSPECTION

PROPER NET HAULING CAPACITY  
IN DAILY SERVICE

ALL WEIGHTS AND TYPES OF LOCOMOTIVES

ON ANY GRADE

From Level to 528 feet per Mile

According to the Actual Records of Practice on Numerous American Railways.

BEING TABLE 170 OF "THE ECONOMIC THEORY OF THE LOCATION OF RAILWAYS," BY A. M. WELLINGTON, M. AM. SOC. C. E.

Correct to nearest ton for 1-4 adhesion and 8 lbs. rolling friction on tangents, even half-tons being dropped. These units are used as those giving results most nearly corresponding with the actual work of locomotives in practice. Higher adhesion, up to 1-3 the weight on drivers, is realized in tests, but not in daily work. Lower rolling friction, down to 3 or 4 lbs. per ton, is realized occasionally in the same way, but the engines cannot be loaded correspondingly and do their daily work.

Applicable to either long or short tons, or any other unit, if the weights of engine be supposed to be given in tons of the same kind. The table gives simply, in effect, the ratio of net load behind engine to the total adhesion, assumed at 1/4 the weight on drivers, for each even ton of adhesion, or each 4 tons on drivers. Intermediate weights can be interpolated by inspection. Adding total weight (as assumed) of engine and tender given in the heading gives the gross weight of engine and train which an engine of any pattern whatsoever can take up any grade with 1/4 adhesion. For 1-3 adhesion the gross load will be 33 1/3 per cent. greater, and for 1-5 adhesion 20 per cent. less. The net load varies slightly with the pattern of engine. For tank engines having any given weight on drivers, correct the table by the difference between its actual weight in service, and that assumed for an engine with tender, with same load on drivers, in preparing this table.

The table gives the fair working capacity for locomotives in every-day service on de-facto grades of the given rate, as evinced by the daily practice of many lines (see Table 138 of the volume above referred to). In single tests they will run some 20 per cent. higher; in winter weather, about 10 per cent. lower. Otherwise any considerable excess in reported loads above the following table indicates simply that the grades are not in reality as high as reported, but are probably operated as momentum grades; and any considerable deficiency indicates either carelessness in loading engines to their capacity, or that the profile grades are in effect increased by unreduced curvature or stopping-points on the maximum grade. Thus, a de-facto level grade for operating purposes hardly exists in the world; nor can it, except with a very unequal traffic enabling all curves and stations to be on a descending grade without impeding up traffic.

PASSENGER AND HIGH-SPEED TRAIN LOADS

Vary greatly from those of the table. The probable maximum resistance on a level at various velocities is given in the following table (the first part of Table 166 of the "Economic Theory of the Location of Railways"). These resistances are probably one-third too high for high-speed work under favorable conditions, but approximate very closely to the ordinary working maximum for which trains are loaded.

TRAIN RESISTANCE ON A LEVEL AS AFFECTED BY VELOCITY.

Freight Trains. HEAVY CONSOLIDATION ENGINE.	TOTAL WEIGHT OF TRAIN		EQUATION OF RESISTANCE, Per Short Ton.	RESISTANCE PER SHORT TON, FOR VELOCITIES, Miles Per Hour.				
	Long Tons.	Short Tons.		10	15	20	25	30
Engine only	70	78.4	$4.82 + .0428V^2$	9.10	14.47	21.94	31.59	43.34
" and 10 loaded cars....	270	302.4	" + .0151V <sup>2</sup>	6.33	8.22	10.86	14.26	18.41
" " 20 " " " " " "	470	526.4	" + .0109V <sup>2</sup>	5.91	7.17	9.18	11.53	14.63
" " 30 " " " " " "	670	750.4	" + .00928V <sup>2</sup>	5.75	6.91	8.53	10.62	13.17
" " 40 " " " " " "	870	974.4	" + .00837V <sup>2</sup>	5.66	6.70	8.17	10.05	12.35
" " 50 " " " " " "	1070	1198.4	" + .0078V <sup>2</sup>	5.60	6.58	7.94	9.70	11.84
" " 75 " " " " " "	1570	1758.4	" + .00703V <sup>2</sup>	5.52	6.40	7.63	9.21	11.15
" " 100 " " " " " "	2070	2318.4	" + .00653V <sup>2</sup>	5.47	6.29	7.33	8.80	10.70

For formulae of resistances for trains of flat cars, subtract about .0012 from coefficient of V<sup>2</sup>. For resistances and formulae per long ton, add 12 per cent.

Passenger Trains. 17 X 24 AMERICAN ENGINE.	TOTAL WEIGHT OF TRAIN.		EQUATION OF RESISTANCE, Per Short Ton.	RESISTANCE PER SHORT TON, FOR VELOCITIES, Miles Per Hour.							
	Long Tons.	Short Tons.		15	20	25	30	40	50	60	70
Engine only	50	56	$4.82 + .03214V^2$	12.05	17.68	24.91	33.75	56.25	85.18	120.54	162.32
" and 2 cars.	100	112	" + .01875V <sup>2</sup>	9.04	12.32	16.54	21.70	34.82	51.70	72.32	96.70
" " 4 " " " " " "	150	168	" + .0143V <sup>2</sup>	8.04	10.54	13.75	17.68	27.60	40.53	56.25	74.82
" " 8 " " " " " "	250	280	" + .0107V <sup>2</sup>	7.23	9.11	11.52	14.46	21.96	31.61	42.30	57.32
" " 12 " " " " " "	350	392	" + .00918V <sup>2</sup>	6.80	8.49	10.56	13.09	19.51	27.78	37.88	49.82
" " 16 " " " " " "	450	504	" + .0083V <sup>2</sup>	6.70	8.16	10.03	12.32	18.15	25.65	34.82	45.65

For resistances and formulae per long ton, add 12 per cent. Weight of cars taken at 25 long tons, 56,000 lbs. each, loaded. Grades, however steep, affect fast trains very slightly unless long, or with stations on them or at their foot. Ordinary undulations not over 50 or 60 ft. high are virtually eliminated by small fluctuations in high speeds (see "Economic Theory of the Location of Railways," p. 346-375).

The approximate maximum speed which any engine can attain in practice on any long de-facto grade at any speed may be determined as follows:

1. Take from the following short table the weight of train to be hauled.
2. Take from the main table the grade that the engine can haul it on at low speed, and subtract the actual grade from it.
3. Find from the following table the speed which increases the virtual grade by the amount of the difference thus obtained.

INCREASE IN VIRTUAL GRADE PER CENT. DUE TO INCREASE IN VELOCITY ABOVE FREIGHT SPEEDS.

(Abstracted from Table 180 of the "Economic Theory of the Location of Railways.")

KIND OF TRAIN.	WEIGHT OF CARS TONS (2000 lbs.)	VELOCITIES IN MILES PER HOUR.							
		20	25	30	40	50	60	70	
Engine and 2 cars.....	56	0.22	0.43	0.68	1.34	2.18	3.22	4.43	
" " 4 " " " " " "	112	0.12	0.29	0.48	0.98	1.62	2.41	3.34	
" " 8 " " " " " "	224	0.06	0.18	0.33	0.72	1.18	1.72	2.47	
" " 12 " " " " " "	336	0.02	0.13	0.25	0.58	0.99	1.49	2.09	
" " 16 " " " " " "	448	0.01	0.10	0.22	0.51	0.88	1.34	1.88	

Example 1. Eight-car train on 1.5 per cent grade. Heavy American engine can pull 224 tons at slow speed on 1.72 per cent. grade, or 0.22 per cent. above the actual. This corresponds to but little more than a 25-mile per hour maximum, by the above table

Example 2. Four car train, same grade and engine. Engine can pull 112 tons on 3.12 per cent. grade at slow speed; 3.12 - 1.50 = 1.62, corresponding by above table to 50 miles per hour.

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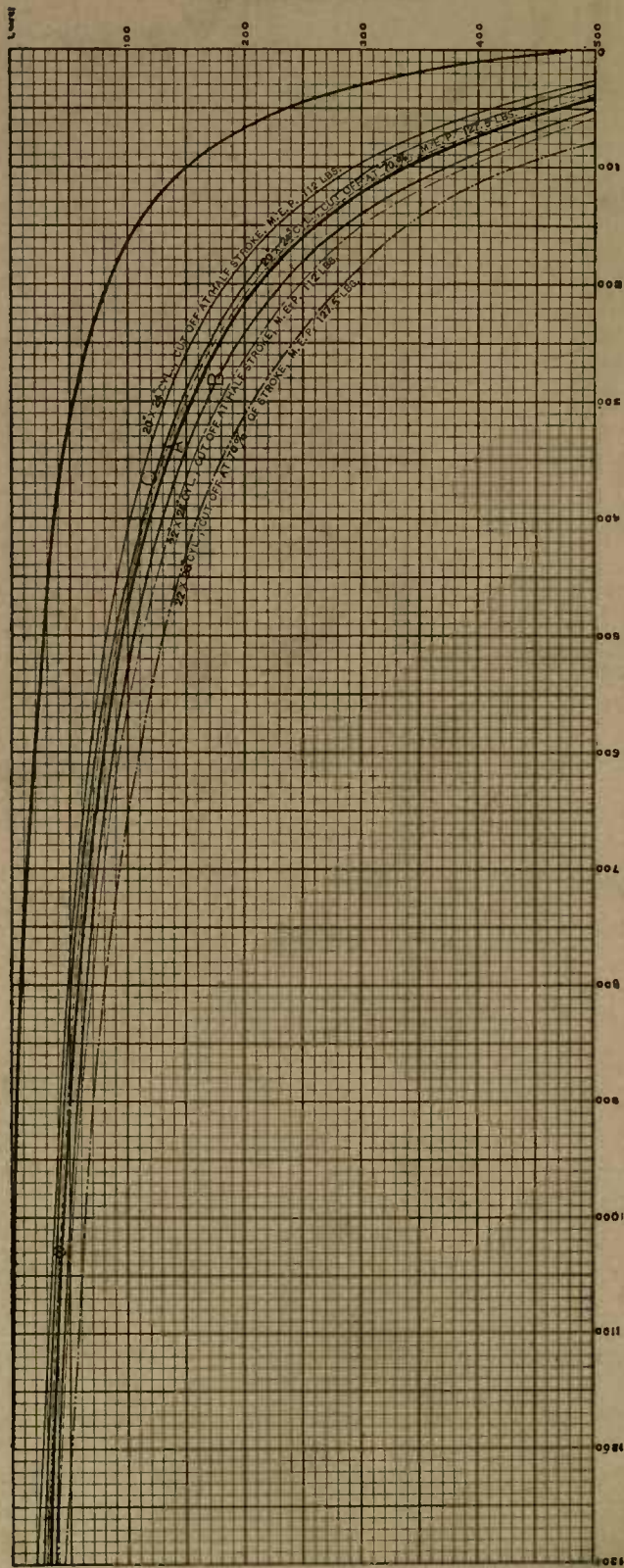


Diagram illustrating the Rate of Variation in the Hauling Power of Engines on Various Grades, and the Ratio of Tractive to Cylinder Power.

NOTES TO DIAGRAM.—This diagram, which constitutes Fig. 169 of "The Economic Theory of the Location of Railways," is reduced from one prepared by Mr. G. W. CUSHING, Supt. of Motive Power, Northern Pacific Ry., with certain lines added. It is somewhat confusing on first examination, from the fact that it is not constructed according to established geometric methods, with a common origin for the two co-ordinate axes at their intersection **O**. It can best be studied by regarding the end of the diagram to the right, as the horizontal axis (axis of *x*), and regarding the curves as indicating the increase of net load from progressive diminutions of grade resistance. It is explained further in the discussion of "The Hauling Power of Engines."

Regarding this end of the diagram as the base-line (or "axis of *x*") as explained beneath the title to it:

1. The lower heavy line represents the progressive increase of train-load (behind tender) as the grade is reduced, for the *lightest American engine* given in Table 170, which begins at **O** at a grade of 480 ft. per mile, and ends at 1198 tons on a level, just within the limits of the diagram.
2. The upper heavy line, marked **A**, represents the same thing for the *heaviest Mastodon engine* given in Table 170 within, so nearly that it is not in error by more than its own width at any point.

Similar lines for all the other nine engines whose tractive capacities are given in Table 170 within, would fall between these two lines at approximately regular intervals. It has not seemed necessary to plot them.

The remaining lines of the diagram give the cylinder and adhesion tractive powers separately for the three different engines below detailed, as computed by Mr. G. W. CUSHING, Supt. M. P. No. Pac. Ry., on the following assumptions:

Rolling friction,  $6\frac{1}{2}$  lbs. per ton in place of 8 lbs. per ton, as for the table within.

Ratio of adhesion,  $\frac{1}{4}$ , as in the table within.

The difference in the rolling friction makes the train-loads slightly greater than those given in Table 170, especially as a level is approached, but makes no great difference on the higher grades. The three engines are as follows:

EN-GINE.	Cylinders.	Drivers.	Trac. P't in Lbs. per Lb. of Effective Pressure.	WEIGHTS.		
				No. of Drivers	On Drivers Engine.	Tender Loaded
<b>A</b> .....	Inches.	Inches.				
	22 x 26	49	256.8	100,000	112,000	65,000
<b>B</b> .....	22 x 26	49	256.8	110,000	112,000	65,000
<b>C</b> .....	20 x 24	49	196	96,000	108,000	65,000
						177,000
						177,000
						173,000

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THE  
ECONOMIC THEORY  
OF THE  
LOCATION OF RAILWAYS.

in analysis of the Conditions controlling the Laying-out of Railways to Effect the most judicious Expenditure of Capital, by ARTHUR MELLE WELINGTON, M. Am. Soc. C. E., late Prin. Asst. Engr. for Location and Surveys Mexican National Ry., Asst. Gen. Man. in Charge of Location Mexican Central Ry., and Ch. Engr. of the American Line from Vera Cruz to Mexico.

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